



CAT-SIUH-2000(3)
SUPERSEDES CAT-SIUH-98(2)



SARAVEL INDUSTRIAL UNIT HEATERS



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INTRODUCTION

SARAVEL industrial unit heaters can be selected from among 8 different size to match specific hydronic requirements of a variety of industrial applications such as factories, warehouses, garages, hangers, and shipping rooms. Capacities range as standard rating is from 117 to 3043 MBH for hot water units; 147 to 4036 MBH for steam units.

Units are designed for optimum throw and distribution of heated air. Each unit is equipped with universal type cowls which individually rotate to permit altering the direction of air flow with respect to the installation position of the unit. The following design features are incorporated into the construction of SARAVEL unit heaters:

UNIT CASINGS

All casings are constructed of heavy gage galvanized steel sheets finished with air dried enamel.

COILS

Coils are constructed of galvanized steel sheet casings with a choice of seamless steel or copper tubes. For hot water applications with temperatures exceeding 130°C and high pressure steam applications greater than 30 psig, 1/2" seamless steel tubes are available with spiral fins made of aluminum or copper. For hot water and steam applications with normal working temperatures and pressures, 5/8" copper tubes with a choice of aluminum or copper fins in plate form are available. Fin spacing is 8FPI. Steam coils have 1 or 2 tubes rows, and hot water coils have 2 or 3 tubes rows. All coils are rated to ARI 410 Standard.

FANS

All fans are forward curved centrifugal type, designed to secure quiet and efficient operation and are made of galvanized sheet metal. Fan assembly including pulleys, sheaves, and shafts are statically and dynamically balanced.

MOTORS

Motors are 3 phase-380V-50Hz and are available at 1450 RPM or 950 RPM for applications where low noise levels are important. Spark proof motors are also available for applications where flammable and volatile gases might leak into the area.

DAMPERS (OPTIONAL)

Face and bypass dampers are of parallel blade type with damper rods that rotate in plastic bushings for smooth operation and correction resistance.

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The logical procedure to follow for considering the several factors required for satisfactory selection and application of horizontal and vertical Industrial Unit Heaters is given below:

1. The first requirement is to calculate the total BTU per hour heat loss of the building. For complete information on heat loss calculations the ASHRAE Guide is recommended.

2. The next step is to consider these points which can have an important bearing on the quantity and type of unit heaters which should be selected to do the job most effectively: (A) Purpose of the building – factory, store, warehouse, etc. (B) Shape of building. (C) Work areas. (D) Arrangement of machinery and equipment, overhead cranes, etc.

3. In designing the system and considering the above points, select Industrial Unit Heaters with a total heating capacity approximately equal to the calculated heat loss. The number of units to use depends largely on the requirement for close temperature control. In order to provide even distribution and close temperature control in areas where this is desirable for comfort, such as stores, offices and certain factory work areas, a quantity of small capacity units should be considered. In other areas where uniform temperature is not an important factor, such as typical factory and warehouse spaces, large capacity units should be used since this result in a more economical installation.

4. The selection of horizontal or vertical units or a combination of both types will usually be dictated by the shape of the building and arrangement of the equipment.

Horizontal Units (horizontal discharge) are recommended for areas having low ceilings, such as stores, show rooms and hallways, as well as areas with comparatively high ceilings, such as factories and warehouses, provided the units can be mounted at a suitable height so the horizontal flow of air will not be stopped by machinery or other obstructions.

Vertical Units (down blast discharge) are particularly suitable for buildings with high ceilings, and for areas that require the units to be mounted at a comparatively high position in order to clear obstacles such as cranes, and high stacks and rows of materials.

5. After selecting the proper quantity and type of units to provide the required heat, certain general rules can then be followed to accomplish the most effective application. (A) Horizontal units should be positioned to blow along the outside walls to obtain a wiping action over the exposed surfaces. Most effective coverage will result if the units can be arranged so the air from one unit blows toward the air intake on another unit, thereby setting up a gentle circulation within the building. This arrangement promotes even distribution and uniform temperature. If additional units are required to handle the heat loss, they can be arranged to blow toward the interior of the building. (B) Vertical units should be located at sufficient height so that the warm air that tends to accumulate near the ceiling will be reirculated to the working level. When only vertical units are used, they should be arranged to provide a blanketing effect over the exposed surfaces and doorways.



SELECTION EXAMPLES

STEAM APPLICATION

An application calls for the selection of an Industrial Unit Heater given the following design conditions:

Entering Air Temperature, (EAT)30°F*
Steam Pressure available15 psig
Heating Load700,000 BTU/Hr

Standard ratings for SARAVEL Industrial Unit Heaters are based on 30 psig steam pressure and 60°F entering air temperature. Since the given conditions are not based on the standard conditions, a correction factor should be applied to the heating load. From [TABLE 3](#) the correction factor for 30°F entering air temperature and 15 psig steam pressure is 1.026. Applying the correction factor:

$$700,000 \times 1.026 = 718,200 \text{ BTU/Hr}$$

From [TABLE 1](#) a two row steam industrial unit heater, Model 2B14S with the following ratings can be selected:

$$\begin{aligned} \text{BTUH} &= 752,000 \\ \text{CFM} &= 7300 \end{aligned}$$

The rating of the selected unit should now be corrected to the job design condition by applying the correction factor:

$$\frac{752,000}{1.026} = 732,900 \text{ BTUH}$$

Since the standard ratings are based on 60°F entering air temperature, in order to obtain the CFM delivery in this case for 30°F entering air, a CFM correction factor should be applied. Enter [TABLE 5](#) at 30°F entering air temperature to read the CFM correction factor:

CFM correction factor from [TABLE 5](#) = 1.060
CFM @ 60°F = 7300 x 1.060 = 7738

The final air temperature, T_{final} , can be obtained using the following equation:

Calculate the final air temperature

The final air temperature can be determined according to the following formula:

$$\begin{aligned} T_{\text{final}} &= \frac{\text{Converted BTU rating}}{\text{CFM @ } 60^\circ \text{ F} \times 1.087} + T_{\text{inlet}} \\ &= \frac{732,900}{7738 \times 1.087} + 30^\circ \text{ F} = 117^\circ \text{ F} \end{aligned}$$

Calculate the amount of condensate formed

In order to size steam traps, the amount of condensate formed per unit should be determined. Multiply the rated capacity of the unit by the steam correction factor found previously and divide by the latent heat of steam (BTU/lb) at the given pressure from [TABLE 9](#).

$$\begin{aligned} \text{Condensate (lb/hr)} &= (752,000 \times 1.026) / 945.5 \\ &= 816 \text{ lb/hr} \end{aligned}$$

HOT WATER APPLICATION

An Industrial Unit Heater is to be selected given the following design conditions:

Entering Air Temperature, (EAT)60°F
Entering Water Temp.,(EWT)210°F
Leaving Water Temp., (LWT)160°F
Heating Load600 MBH

Since the ratings for hot water applications are based on 180°F EWT and this application calls for 210°F EWT, the correction from [TABLE 4](#) at 60°F EAT and 210°F EWT is 1.270. Applying the correction factor to the load:

$$\frac{600,000}{1.270} = 472,000 \text{ BTU/Hr}$$

From [TABLE 2](#) a 3 row Model 2B14 with the following ratings can be selected:

$$\begin{aligned} \text{BTUH} &= 502,900 \\ \text{CFM} &= 6700 \\ \text{GPM} &= 51.7 \end{aligned}$$

The rating of the selected unit should now be corrected to the job design condition by applying the correction factor:

$$502,900 \times 1.270 = 638,000 \text{ BTU/Hr}$$

Calculate the final air temperature

The final air temperature can be obtained by the following equation:

$$\begin{aligned} T_{\text{final}} &= \frac{\text{Converted BTU rating}}{\text{CFM @ } 60^\circ \text{ F} \times 1.087} + T_{\text{inlet}} \\ &= \frac{638,000}{6700 \times 1.087} + 60^\circ \text{ F} = 148^\circ \text{ F} \end{aligned}$$

*Note: This is a full fresh air application.



SELECTION EXAMPLES

Calculate CFM @ the final temperature

The following formula can be used to determine the CFM at the final air temperature:

$$\begin{aligned} \text{CFM @ Final Temp.} &= \text{Rated CFM} \times \frac{460 + T_{\text{final}}}{460 + 70} \\ &= 6700 \times \frac{460+148}{460+70} = 7686 \end{aligned}$$

Determine GPM

$$\begin{aligned} \text{GPM} &= \frac{\text{BTU/Hr}}{500 \Delta T} \\ &= \frac{638,000}{500 \times 50} = 25.5 \end{aligned}$$

where ΔT is the water temperature drop (210 – 160 = 50°F) through the unit.

Calculate pressure drop in the coil

$$\begin{aligned} \% \text{ Water Quantity} &= \frac{\text{GPM}}{\text{Rated GPM} \leftarrow \text{From TABLE 2}} \\ &= \frac{25.5}{51.7} = 49\% \end{aligned}$$

Enter [TABLE 6](#) and interpolate between 40 and 50 percent Water Quantity to obtain the pressure drop multiplier as 0.28. Next apply the multiplier to the rated pressure drop for Model 2B14 in [TABLE 2](#) under the P.D. column:

$$\text{P.D.} = 1.842 \times 0.28 = 0.52 \text{ Feet Water}$$

RPM and HP CORRECTION FACTORS

If ductwork is to be installed to supply heated air to a location other than where the unit is installed, it is than necessary to adjust motor RPM and HP.

Suppose the given external static pressure for model 2B14 in the previous example is:

$$\text{Ext. Sp} = 3/4" \text{ w.g.}$$

From [TABLE 2](#), at 6700 CFM the cowl outlet velocity is 2820 FPM. Entering [TABLE 7](#) at 3/4" w.g. and interpolating between 2800 and 3000 FPM, the

RPM correction factor is calculated as 1.238. Similarly the HP correction factor is calculated as 1.495.

Applying the correction factors to RPM and HP listed in [TABLE 7](#) :

$$\text{RPM} = 743 \times 1.242 = 920$$

$$\text{HP} = 3 \times 1.495 = 4.5 \Rightarrow 5.5 \text{ HP}$$



RATINGS

TABLE 1. STEAM INDUSTRIAL UNIT HEATER RATINGS *

| MODEL | CFM | HP | RPM | 1ROW | | | 2 ROW | | |
|-------|-------|------|------|---------|-------|--------|---------|-------|--------|
| | | | | BTU/Hr | LDB°F | FPM ** | BTU/Hr | LDB°F | FPM ** |
| 1B13S | 2000 | 0.75 | 814 | 146900 | 128 | 1680 | 247600 | 174 | 1810 |
| | 2300 | 0.75 | 837 | 157600 | 123 | 1920 | 269000 | 168 | 2060 |
| | 2600 | 1 | 873 | 167800 | 119 | 2150 | 288900 | 162 | 2310 |
| | 2900 | 1 | 914 | 176800 | 116 | 2390 | 307400 | 158 | 2560 |
| | 3200 | 1.5 | 959 | 185500 | 113 | 2620 | 324900 | 153 | 2810 |
| | 3500 | 1.5 | 1004 | 193700 | 111 | 2860 | 341500 | 150 | 3050 |
| 2B11S | 2750 | 1 | 959 | 203000 | 128 | 1800 | 341900 | 174 | 1950 |
| | 3200 | 1 | 949 | 219200 | 123 | 2080 | 374100 | 168 | 2240 |
| | 3650 | 1.5 | 956 | 234100 | 119 | 2360 | 403800 | 162 | 2530 |
| | 4100 | 1.5 | 982 | 247800 | 116 | 2630 | 431500 | 157 | 2820 |
| | 4550 | 2 | 1021 | 260700 | 113 | 2910 | 457400 | 152 | 3110 |
| | 5000 | 2 | 1069 | 272800 | 110 | 3180 | 481900 | 149 | 3400 |
| 2B13S | 3750 | 1.5 | 827 | 282400 | 129 | 1780 | 473800 | 176 | 1920 |
| | 4300 | 1.5 | 821 | 302800 | 125 | 2030 | 514300 | 170 | 2180 |
| | 4850 | 1.5 | 829 | 321600 | 121 | 2270 | 551900 | 165 | 2440 |
| | 5400 | 2 | 852 | 339200 | 118 | 2520 | 587200 | 160 | 2700 |
| | 5950 | 2 | 886 | 355700 | 115 | 2760 | 620400 | 156 | 2950 |
| | 6500 | 3 | 924 | 371300 | 113 | 3000 | 651800 | 152 | 3210 |
| 2B14S | 5500 | 2 | 720 | 374200 | 123 | 2290 | 639400 | 167 | 2460 |
| | 6100 | 2 | 728 | 394000 | 119 | 2530 | 679100 | 162 | 2710 |
| | 6700 | 3 | 743 | 412700 | 117 | 2760 | 716500 | 158 | 2960 |
| | 7300 | 3 | 766 | 430300 | 114 | 3000 | 752000 | 155 | 3210 |
| | 7900 | 3 | 794 | 447100 | 112 | 3230 | 785900 | 152 | 3450 |
| | 8500 | 4 | 824 | 463200 | 110 | 3460 | 818300 | 149 | 3700 |
| 2B17S | 7000 | 3 | 587 | 492900 | 125 | 1940 | 837200 | 170 | 2090 |
| | 8200 | 3 | 583 | 533500 | 120 | 2250 | 918400 | 163 | 2420 |
| | 9400 | 3 | 592 | 570600 | 116 | 2570 | 992900 | 157 | 2750 |
| | 10600 | 4 | 614 | 605000 | 113 | 2880 | 1062100 | 152 | 3080 |
| | 11800 | 5.5 | 643 | 637000 | 110 | 3190 | 1126800 | 148 | 3400 |
| | 13000 | 5.5 | 675 | 667100 | 107 | 3500 | 1187700 | 144 | 3720 |
| 2B19S | 10000 | 4 | 523 | 719600 | 126 | 2060 | 1217500 | 172 | 2220 |
| | 12000 | 4 | 536 | 788600 | 121 | 2440 | 1355300 | 164 | 2630 |
| | 14000 | 5.5 | 557 | 850900 | 116 | 2830 | 1480300 | 157 | 3030 |
| | 16000 | 7.5 | 587 | 907900 | 112 | 3210 | 1595300 | 152 | 3430 |
| | 18000 | 10 | 623 | 960800 | 109 | 3590 | 1702000 | 147 | 3830 |
| | 20000 | 10 | 693 | 1010200 | 107 | 3970 | 1801900 | 143 | 4230 |
| 3B19S | 15000 | 5.5 | 523 | 1045700 | 124 | 2050 | 1779300 | 169 | 2210 |
| | 17600 | 5.5 | 530 | 1132500 | 119 | 2380 | 1953100 | 162 | 2560 |
| | 20200 | 7.5 | 548 | 1211900 | 115 | 2720 | 2112500 | 156 | 2910 |
| | 22800 | 10 | 571 | 1285200 | 112 | 3050 | 2260300 | 151 | 3260 |
| | 25400 | 10 | 597 | 1353600 | 109 | 3380 | 2398400 | 147 | 3600 |
| | 28000 | 15 | 625 | 1417900 | 107 | 3710 | 2528300 | 143 | 3950 |
| 4B19S | 24000 | 7.5 | 532 | 1733400 | 126 | 2470 | 2930800 | 172 | 2660 |
| | 27600 | 10 | 552 | 1859800 | 122 | 2820 | 3183000 | 166 | 3030 |
| | 31200 | 15 | 577 | 1976500 | 118 | 3160 | 3417000 | 161 | 3400 |
| | 34800 | 15 | 604 | 2085300 | 115 | 3510 | 3635600 | 156 | 3760 |
| | 38400 | 20 | 634 | 2187400 | 112 | 3850 | 3841400 | 152 | 4120 |
| | 42000 | 25 | 700 | 2283900 | 110 | 4200 | 4036000 | 148 | 4480 |

* Ratings are based on 30 Psig Steam, 60°F Entering Air Temp. and 5/8" copper tubes with plate fins as standard.

** FPM is discharge air velocity (ft/min), at Final Air Temp. equivalent to cowl outlet velocity.

Note: a) Fan ratings are based on standard air (Density=0.075 lb/ft³ at sea level, 70°F, and 29.92 inches of mercury barometric pressure) with total static pressure of 1" w.g.

b) All coils are rated to ARI 410 Standard.

c) Fin spacing is 8FPI.



RATINGS

TABLE 2. HOT WATER INDUSTRIAL UNIT HEATER RATINGS*

| MODEL | CFM | HP | RPM | 2 ROWS | | | | | 3 ROWS | | | | |
|-------|-------|------|------|---------|-------|------------|-----------|-------|---------|-------|------------|-----------|-------|
| | | | | BTU/Hr | GPM | PD (ft) | LDB °F | FPM** | BTU/Hr | GPM | PD (ft) | LDB °F | FPM** |
| 1B13 | 2000 | 0.75 | 814 | 117100 | 12.0 | 0.096 | 114 | 1640 | 158900 | 16.3 | 0.249 | 133 | 1700 |
| | 2300 | 0.75 | 837 | 127800 | 13.1 | 0.112 | 111 | 1880 | 175100 | 18.0 | 0.296 | 130 | 1940 |
| | 2600 | 1 | 873 | 137700 | 14.2 | 0.128 | 109 | 2110 | 190400 | 19.6 | 0.345 | 127 | 2180 |
| | 2900 | 1 | 914 | 147100 | 15.1 | 0.144 | 107 | 2350 | 204800 | 21.0 | 0.394 | 125 | 2420 |
| | 3200 | 1.5 | 959 | 156000 | 16.0 | 0.160 | 105 | 2580 | 218600 | 22.5 | 0.443 | 123 | 2670 |
| | 3500 | 1.5 | 1004 | 164400 | 16.9 | 0.176 | 103 | 2820 | 231800 | 23.8 | 0.492 | 121 | 2910 |
| 2B11 | 2750 | 1 | 959 | 168000 | 17.3 | 0.188 | 116 | 1770 | 224700 | 23.1 | 0.477 | 135 | 1830 |
| | 3200 | 1 | 949 | 184900 | 19.0 | 0.224 | 113 | 2050 | 249900 | 25.7 | 0.578 | 132 | 2110 |
| | 3650 | 1.5 | 956 | 200500 | 20.6 | 0.259 | 111 | 2320 | 273500 | 28.1 | 0.681 | 129 | 2400 |
| | 4100 | 1.5 | 982 | 215200 | 22.1 | 0.294 | 108 | 2600 | 295900 | 30.4 | 0.785 | 126 | 2680 |
| | 4550 | 2 | 1021 | 229100 | 23.5 | 0.330 | 106 | 2880 | 317000 | 32.6 | 0.889 | 124 | 2970 |
| | 5000 | 2 | 1069 | 242300 | 24.9 | 0.365 | 105 | 3150 | 337300 | 34.7 | 0.994 | 122 | 3250 |
| 2B13 | 3750 | 1.5 | 827 | 237300 | 24.4 | 0.281 | 118 | 1750 | 314100 | 32.3 | 0.699 | 137 | 1810 |
| | 4300 | 1.5 | 821 | 259000 | 26.6 | 0.329 | 115 | 2000 | 346100 | 35.6 | 0.833 | 134 | 2060 |
| | 4850 | 1.5 | 829 | 279300 | 28.7 | 0.377 | 113 | 2240 | 376400 | 38.7 | 0.969 | 131 | 2310 |
| | 5400 | 2 | 852 | 298400 | 30.7 | 0.425 | 111 | 2490 | 405100 | 41.6 | 1.107 | 129 | 2560 |
| | 5950 | 2 | 886 | 316600 | 32.5 | 0.473 | 109 | 2730 | 432500 | 44.4 | 1.246 | 127 | 2820 |
| | 6500 | 3 | 924 | 333900 | 34.3 | 0.521 | 107 | 2970 | 458800 | 47.1 | 1.386 | 125 | 3070 |
| 2B14 | 5500 | 2 | 720 | 329000 | 33.8 | 0.571 | 115 | 2260 | 439000 | 45.1 | 1.441 | 133 | 2330 |
| | 6100 | 2 | 728 | 351100 | 36.1 | 0.641 | 113 | 2500 | 471700 | 48.5 | 1.641 | 131 | 2580 |
| | 6700 | 3 | 743 | 372000 | 38.2 | 0.712 | 111 | 2730 | 502900 | 51.7 | 1.842 | 129 | 2820 |
| | 7300 | 3 | 766 | 392000 | 40.3 | 0.783 | 109 | 2970 | 532900 | 54.7 | 2.045 | 127 | 3060 |
| | 7900 | 3 | 794 | 411100 | 42.2 | 0.853 | 108 | 3210 | 561700 | 57.7 | 2.249 | 125 | 3310 |
| | 8500 | 4 | 824 | 429500 | 44.1 | 0.923 | 107 | 3440 | 589400 | 60.6 | 2.454 | 124 | 3550 |
| 2B17 | 7000 | 3 | 587 | 437500 | 45.0 | 0.913 | 118 | 1920 | 577300 | 59.3 | 2.259 | 136 | 1980 |
| | 8200 | 3 | 583 | 483500 | 49.7 | 1.093 | 114 | 2230 | 644900 | 66.3 | 2.759 | 132 | 2300 |
| | 9400 | 3 | 592 | 526200 | 54.1 | 1.274 | 112 | 2550 | 708200 | 72.8 | 3.268 | 129 | 2630 |
| | 10600 | 4 | 614 | 566200 | 58.2 | 1.454 | 109 | 2860 | 767800 | 78.9 | 3.782 | 127 | 2950 |
| | 11800 | 5.5 | 643 | 603900 | 62.0 | 1.634 | 107 | 3170 | 824300 | 84.7 | 4.299 | 124 | 3270 |
| | 13000 | 5.5 | 675 | 639500 | 65.7 | 1.812 | 105 | 3480 | 878000 | 90.2 | 4.819 | 122 | 3590 |
| 2B19 | 10000 | 4 | 523 | 640000 | 65.8 | 1.136 | 119 | 2030 | 839600 | 86.3 | 2.784 | 137 | 2090 |
| | 12000 | 4 | 536 | 718700 | 73.8 | 1.401 | 115 | 2420 | 954600 | 98.1 | 3.511 | 133 | 2500 |
| | 14000 | 5.5 | 557 | 791000 | 81.3 | 1.666 | 112 | 2810 | 1061400 | 109.0 | 4.252 | 130 | 2900 |
| | 16000 | 7.5 | 587 | 858200 | 88.2 | 1.931 | 109 | 3190 | 1161300 | 119.3 | 5.002 | 127 | 3290 |
| | 18000 | 10 | 623 | 921100 | 94.6 | 2.194 | 107 | 3580 | 1255300 | 129.0 | 5.758 | 124 | 3690 |
| | 20000 | 10 | 693 | 980300 | 100.7 | 2.455 | 105 | 3960 | 1344400 | 138.1 | 6.517 | 122 | 4080 |
| 3B19 | 15000 | 5.5 | 523 | 959400 | 98.6 | 3.081 | 119 | 2030 | 1254500 | 128.9 | 7.503 | 137 | 2090 |
| | 17600 | 5.5 | 530 | 1062000 | 109.1 | 3.702 | 116 | 2370 | 1403600 | 144.2 | 9.191 | 133 | 2440 |
| | 20200 | 7.5 | 548 | 1157200 | 118.9 | 4.323 | 113 | 2700 | 1543100 | 158.5 | 10.907 | 130 | 2790 |
| | 22800 | 10 | 571 | 1246300 | 128.0 | 4.943 | 110 | 3040 | 1674500 | 172.0 | 12.643 | 128 | 3130 |
| | 25400 | 10 | 597 | 1330200 | 136.7 | 5.561 | 108 | 3370 | 1798900 | 184.8 | 14.391 | 125 | 3480 |
| | 28000 | 15 | 625 | 1409700 | 144.8 | 6.175 | 106 | 3710 | 1917200 | 197.0 | 16.147 | 123 | 3820 |
| 4B19 | 24000 | 7.5 | 532 | 1594800 | 163.8 | 5.660 | 121 | 2450 | 2066000 | 212.3 | 13.554 | 139 | 2520 |
| | 27600 | 10 | 552 | 1745200 | 179.3 | 6.661 | 118 | 2800 | 2282400 | 234.5 | 16.228 | 136 | 2880 |
| | 31200 | 15 | 577 | 1886200 | 193.8 | 7.665 | 116 | 3150 | 2486800 | 255.5 | 18.948 | 133 | 3250 |
| | 34800 | 15 | 604 | 2019200 | 207.4 | 8.670 | 113 | 3500 | 2680800 | 275.4 | 21.702 | 131 | 3610 |
| | 38400 | 20 | 634 | 2145300 | 220.4 | 9.673 | 111 | 3850 | 2865700 | 294.4 | 24.483 | 129 | 3960 |
| | 42000 | 25 | 700 | 2265400 | 232.7 | 10.673 | 110 | 4190 | 3042600 | 312.6 | 27.281 | 127 | 4320 |

* Ratings are based on 180°F entering water temp. & 160°F leaving water temp. & 60°F entering air temp. and 5/8" copper tubes with plate fins as standard.

** FPM is discharge air velocity (ft/min), at Final Air Temp. equivalent to cowl outlet velocity.

Note: a) Fan ratings are based on standard air (Density=0.075 lb/ft³ at sea level, 70°F, and 29.92 inches of mercury barometric pressure) with total static pressure of 1" w.g.

b) All coils are rated to ARI 410 Standard.

c) Fin spacing is 8FPI.

d) Shaded areas are conditions where water velocities exceed 6 fps.



CORRECTION FACTORS

TABLE 3. STEAM CORRECTION FACTORS*

| ENT. AIR TEMP °F | STEAM PRESSURE IN POUNDS PER SQUARE INCH - PSIG | | | | | | | | | | | | | | | |
|------------------|---|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| | 0 | 2 | 5 | 10 | 15 | 20 | 30 | 40 | 50 | 60 | 80 | 100 | 125 | 150 | 175 | 200 |
| -30 | 1.133 | 1.163 | 1.200 | 1.258 | 1.308 | 1.348 | 1.420 | 1.482 | 1.532 | 1.585 | 1.654 | 1.717 | 1.792 | 1.847 | 1.903 | 1.956 |
| -20 | 1.082 | 1.113 | 1.153 | 1.211 | 1.258 | 1.301 | 1.373 | 1.431 | 1.483 | 1.528 | 1.605 | 1.670 | 1.740 | 1.801 | 1.855 | 1.903 |
| -10 | 1.036 | 1.066 | 1.107 | 1.164 | 1.212 | 1.254 | 1.325 | 1.384 | 1.436 | 1.481 | 1.558 | 1.623 | 1.693 | 1.755 | 1.808 | 1.856 |
| 0 | 0.989 | 1.020 | 1.060 | 1.117 | 1.166 | 1.207 | 1.278 | 1.338 | 1.389 | 1.434 | 1.512 | 1.576 | 1.647 | 1.708 | 1.762 | 1.810 |
| 10 | 0.942 | 0.973 | 1.013 | 1.071 | 1.118 | 1.161 | 1.233 | 1.292 | 1.342 | 1.388 | 1.465 | 1.530 | 1.601 | 1.660 | 1.715 | 1.764 |
| 20 | 0.896 | 0.926 | 0.967 | 1.024 | 1.073 | 1.114 | 1.186 | 1.244 | 1.296 | 1.341 | 1.418 | 1.483 | 1.553 | 1.615 | 1.669 | 1.717 |
| 30 | 0.849 | 0.880 | 0.920 | 0.977 | 1.026 | 1.067 | 1.139 | 1.198 | 1.250 | 1.294 | 1.372 | 1.436 | 1.506 | 1.568 | 1.622 | 1.670 |
| 40 | 0.802 | 0.833 | 0.873 | 0.930 | 0.978 | 1.021 | 1.092 | 1.151 | 1.202 | 1.248 | 1.325 | 1.390 | 1.461 | 1.521 | 1.575 | 1.628 |
| 45 | 0.779 | 0.810 | 0.850 | 0.907 | 0.955 | 0.997 | 1.069 | 1.128 | 1.180 | 1.224 | 1.302 | 1.366 | 1.436 | 1.496 | 1.552 | 1.601 |
| 50 | 0.756 | 0.786 | 0.827 | 0.884 | 0.932 | 0.974 | 1.045 | 1.104 | 1.156 | 1.201 | 1.273 | 1.343 | 1.414 | 1.474 | 1.529 | 1.576 |
| 55 | 0.732 | 0.763 | 0.803 | 0.861 | 0.908 | 0.951 | 1.023 | 1.081 | 1.133 | 1.178 | 1.255 | 1.320 | 1.390 | 1.451 | 1.505 | 1.553 |
| 60 | 0.709 | 0.740 | 0.780 | 0.837 | 0.885 | 0.927 | 1.000 | 1.058 | 1.109 | 1.154 | 1.231 | 1.297 | 1.367 | 1.427 | 1.482 | 1.531 |
| 65 | 0.686 | 0.716 | 0.757 | 0.814 | 0.862 | 0.904 | 0.976 | 1.034 | 1.086 | 1.131 | 1.209 | 1.273 | 1.343 | 1.407 | 1.459 | 1.506 |
| 70 | 0.662 | 0.693 | 0.733 | 0.791 | 0.838 | 0.881 | 0.935 | 1.011 | 1.063 | 1.108 | 1.186 | 1.250 | 1.320 | 1.380 | 1.435 | 1.484 |
| 75 | 0.639 | 0.670 | 0.710 | 0.767 | 0.815 | 0.857 | 0.930 | 0.988 | 1.040 | 1.084 | 1.163 | 1.226 | 1.297 | 1.357 | 1.412 | 1.460 |
| 80 | 0.616 | 0.646 | 0.687 | 0.744 | 0.792 | 0.834 | 0.906 | 0.965 | 1.016 | 1.061 | 1.139 | 1.203 | 1.273 | 1.335 | 1.389 | 1.436 |
| 85 | 0.592 | 0.623 | 0.663 | 0.720 | 0.768 | 0.811 | 0.883 | 0.941 | 0.993 | 1.038 | 1.116 | 1.180 | 1.251 | 1.310 | 1.365 | 1.414 |
| 90 | 0.569 | 0.600 | 0.640 | 0.696 | 0.745 | 0.787 | 0.860 | 0.918 | 0.969 | 1.014 | 1.093 | 1.156 | 1.226 | 1.288 | 1.342 | 1.390 |
| 100 | 0.522 | 0.553 | 0.593 | 0.650 | 0.698 | 0.732 | 0.813 | 0.871 | 0.923 | 0.968 | 1.045 | 1.110 | 1.181 | 1.240 | 1.295 | 1.344 |
| 110 | 0.476 | 0.506 | 0.547 | 0.603 | 0.652 | 0.694 | 0.766 | 0.825 | 0.876 | 0.921 | 0.998 | 1.063 | 1.134 | 1.194 | 1.248 | 1.297 |
| 120 | 0.429 | 0.460 | 0.500 | 0.556 | 0.605 | 0.647 | 0.720 | 0.778 | 0.830 | 0.874 | 0.952 | 1.027 | 1.086 | 1.147 | 1.201 | 1.251 |
| 140 | 0.336 | 0.366 | 0.407 | 0.464 | 0.512 | 0.554 | 0.626 | 0.685 | 0.737 | 0.781 | 0.858 | 0.923 | 0.993 | 1.055 | 1.108 | 1.156 |
| 160 | 0.242 | 0.273 | 0.313 | 0.370 | 0.418 | 0.460 | 0.533 | 0.591 | 0.642 | 0.688 | 0.765 | 0.831 | 0.901 | 0.961 | 1.012 | 1.065 |
| 180 | 0.149 | 0.179 | 0.220 | 0.277 | 0.325 | 0.367 | 0.439 | 0.498 | 0.550 | 0.594 | 0.671 | 0.737 | 0.808 | 0.868 | 0.921 | 0.970 |
| 200 | 0.056 | 0.085 | 0.127 | 0.183 | 0.232 | 0.274 | 0.345 | 0.405 | 0.455 | 0.501 | 0.577 | 0.643 | 0.713 | 0.775 | 0.829 | 0.876 |

*Standard ratings are based on 30 psig steam pressure @ 60°F Entering Air Temperature and 5/8" copper tubes with plate fins.
 For steam pressure greater than 30 psig, 1/2" seamless steel tubes with spiral fins are available.
 Corrected Load = Rating from TABLE 1 x Correction factor from TABLE 3

TABLE 4. HOT WATER CORRECTION FACTORS

| ENTERING AIR TEMPERATURE °F | ENTERING WATER TEMPERATURE °F | | | | | | | | | | |
|-----------------------------|-------------------------------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| | 150 | 160 | 170 | 180 | 190 | 200 | 210 | 220 | 230 | 240 | 250 |
| 30 | 0.980 | 1.070 | 1.160 | 1.250 | 1.340 | 1.430 | 1.525 | 1.615 | 1.710 | 1.800 | 1.895 |
| 40 | 0.900 | 0.985 | 1.075 | 1.165 | 1.255 | 1.350 | 1.440 | 1.530 | 1.625 | 1.715 | 1.810 |
| 50 | 0.815 | 0.905 | 0.995 | 1.085 | 1.175 | 1.265 | 1.355 | 1.445 | 1.540 | 1.630 | 1.720 |
| 60 | 0.735 | 0.825 | 0.910 | 1.000 | 1.090 | 1.180 | 1.270 | 1.360 | 1.455 | 1.545 | 1.635 |
| 70 | 0.655 | 0.740 | 0.830 | 0.915 | 1.005 | 1.095 | 1.185 | 1.275 | 1.365 | 1.460 | 1.550 |
| 80 | 0.570 | 0.660 | 0.745 | 0.840 | 0.920 | 1.010 | 1.100 | 1.190 | 1.280 | 1.370 | 1.465 |
| 90 | 0.490 | 0.575 | 0.660 | 0.750 | 0.840 | 0.925 | 1.015 | 1.105 | 1.195 | 1.285 | 1.380 |
| 100 | 0.410 | 0.495 | 0.580 | 0.665 | 0.755 | 0.840 | 0.930 | 1.020 | 1.110 | 1.200 | 1.290 |

TABLE 5. CFM CORRECTION FACTORS

| ENTERING WATER TEMPERATURE °F | | | | | | | | | | | |
|-------------------------------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| -10 | 0 | 10 | 20 | 30 | 40 | 50 | 60 | 70 | 80 | 90 | 100 |
| 1.155 | 1.130 | 1.105 | 1.082 | 1.060 | 1.040 | 1.020 | 1.000 | 0.982 | 0.964 | 0.945 | 0.930 |



CORRECTION FACTORS

TABLE 6. COIL PRESSURE DROP CORRECTION FACTORS *

| PERCENT WATER QUANTITY G.P.M. | | | | | | | | | | | | | | |
|-------------------------------|------|------|------|------|------|------|------|------|------|------|------|------|------|------|
| 40 | 50 | 60 | 70 | 80 | 90 | 100 | 110 | 120 | 130 | 140 | 150 | 160 | 170 | 180 |
| 0.19 | 0.29 | 0.40 | 0.53 | 0.67 | 0.83 | 1.00 | 1.18 | 1.38 | 1.59 | 1.81 | 2.05 | 2.30 | 2.55 | 2.83 |

* Corrected Coil Pressure Drop = P.D. from TABLE 2 x Correction factor from TABLE 6

TABLE 7. RPM CORRECTION FACTORS

| COWEL OUTLET VEL. (FPM) | EXTERNAL STATIC PRESSURE (Inches Water Gage) | | | | |
|----------------------------|---|------|------|------|-------|
| | 1/4 | 1/2 | 3/4 | 1 | 1 1/4 |
| 1800 | 1.16 | 1.31 | 1.45 | | |
| 2000 | 1.15 | 1.30 | 1.44 | | |
| 2200 | 1.14 | 1.28 | 1.42 | 1.53 | 1.65 |
| 2400 | 1.12 | 1.24 | 1.36 | 1.47 | 1.58 |
| 2600 | 1.11 | 1.21 | 1.30 | 1.40 | 1.50 |
| 2800 | 1.07 | 1.15 | 1.24 | 1.33 | 1.42 |
| 3000 | 1.07 | 1.14 | 1.22 | 1.30 | 1.39 |
| 3200 | 1.06 | 1.13 | 1.20 | 1.28 | 1.35 |
| 3400 | 1.06 | 1.12 | 1.18 | 1.24 | 1.30 |
| 3600 | 1.05 | 1.10 | 1.15 | 1.19 | 1.24 |
| 3800 | 1.05 | 1.09 | 1.14 | 1.18 | 1.22 |

NOTE: The RPM correction (At various external static pressures) must be made by multiplying its respective ratings capacity table values by the factors in the TABLE 7.

TABLE 8. HP CORRECTION FACTORS

| COWEL OUTLET VEL. (FPM) | EXTERNAL STATIC PRESSURE (Inches Water Gage) | | | | |
|----------------------------|---|------|------|------|-------|
| | 1/4 | 1/2 | 3/4 | 1 | 1 1/4 |
| 1800 | 1.35 | 1.72 | 2.09 | | |
| 2000 | 1.32 | 1.68 | 2.13 | | |
| 2200 | 1.29 | 1.63 | 1.98 | 2.34 | 2.71 |
| 2400 | 1.24 | 1.52 | 1.82 | 2.14 | 2.46 |
| 2600 | 1.19 | 1.41 | 1.66 | 1.93 | 2.21 |
| 2800 | 1.14 | 1.31 | 1.50 | 1.73 | 1.96 |
| 3000 | 1.13 | 1.29 | 1.45 | 1.65 | 1.86 |
| 3200 | 1.13 | 1.26 | 1.41 | 1.59 | 1.77 |
| 3400 | 1.11 | 1.23 | 1.36 | 1.50 | 1.65 |
| 3600 | 1.11 | 1.21 | 1.31 | 1.40 | 1.52 |
| 3800 | 1.10 | 1.20 | 1.29 | 1.39 | 1.50 |

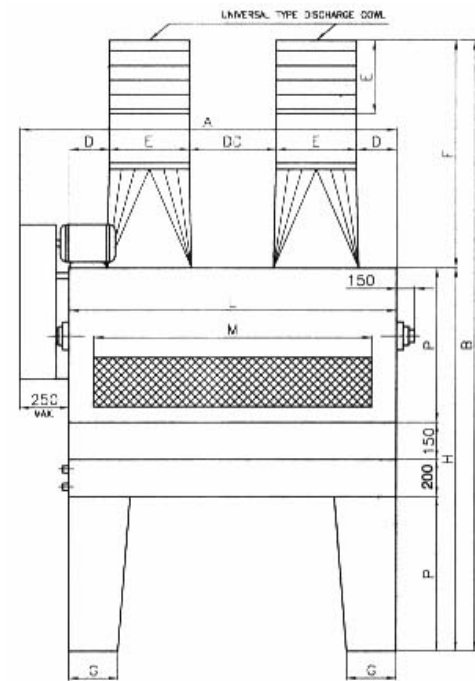
NOTE: The HP correction (At various external static pressures) must be made by multiplying its respective ratings capacity table values by the factors in the TABLE 8.



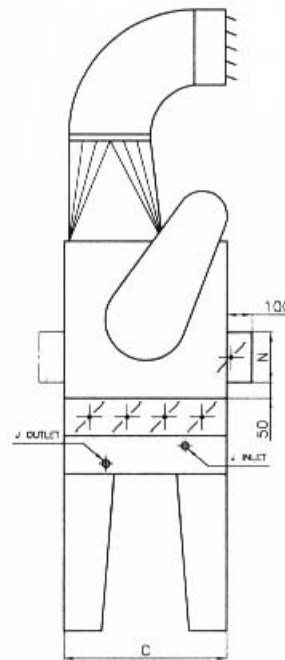
DIMENSIONS

TABLE 11. INDUSTRIAL UNIT HEATER DIMENSIONS

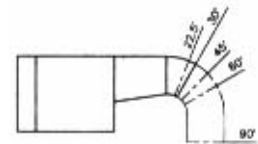
| MODEL | A | B | C | D | DD | E | F | G | H | J | K | L | M | N | P | No. of Outlets |
|-------|------|------|------|-----|------|-----|------|-----|------|-------|-----|------|------|-----|------|----------------|
| 1B13 | 1150 | 2600 | 600 | 275 | ---- | 330 | 950 | 200 | 1600 | 1.25" | 500 | 900 | 650 | 200 | 650 | 1 |
| 2B11 | 1400 | 2600 | 650 | 150 | 290 | 280 | 950 | 200 | 1600 | 1.5" | 500 | 1150 | 1000 | 200 | 650 | 2 |
| 2B13 | 1600 | 2600 | 750 | 180 | 330 | 280 | 950 | 200 | 1600 | 1.5" | 500 | 1350 | 1200 | 200 | 650 | 2 |
| 2B14 | 1850 | 2750 | 750 | 250 | 400 | 350 | 1000 | 200 | 1700 | 2.0" | 550 | 1600 | 1400 | 200 | 700 | 2 |
| 2B17 | 2200 | 2950 | 850 | 280 | 530 | 430 | 1100 | 200 | 1800 | 2.0" | 550 | 1950 | 1750 | 300 | 750 | 2 |
| 2B19 | 2400 | 3650 | 1100 | 300 | 550 | 500 | 1300 | 300 | 2300 | 2.5" | 600 | 2150 | 1950 | 300 | 1000 | 2 |
| 3B19 | 3250 | 3650 | 1100 | 250 | 500 | 500 | 1300 | 300 | 2300 | 2.5" | 600 | 3000 | 2800 | 300 | 1000 | 3 |
| 4B19 | 4100 | 3650 | 1450 | 250 | 450 | 500 | 1300 | 350 | 2300 | 2.5" | 600 | 3850 | 3600 | 300 | 1000 | 4 |



UP BLAST DISCHARGE
WITH FREE STANDING LEGS (OR SUSPENDED)



UP BLAST DISCHARGE (SUSPENDED)

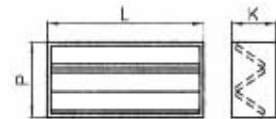


TOP HORIZONTAL DISCHARGE (SUSPENDED)



DOWN BLAST DISCHARGE (SUSPENDED)

CHOOSE DESIRED DISCHARGE ANGLE FROM ABOVE.



FILTER BOX

THE FILTER BOX CAN BE USED ON ALL ARRANGEMENTS (OPTIONAL)

FIGURE 2. INDUSTRIAL UNIT HEATER DIMENSIONS
Drive, face and bypass damper shaft, coil connections and filter access door may be on either end.

Piping Suggestions

As the function of a unit heater is to transfer heat from steam or hot water to the surrounding space, it is necessary that the steam or hot water be delivered to the unit and removed from it, in the required quantity and condition. The piping of the unit heaters must conform strictly to the system requirements, while at the same time permitting the heaters to function as intended. The following are a few piping guidelines which must be observed when designing piping systems for unit heaters.

Piping size should be adequate to handle both steam and condensate, under the maximum load condition. With steam unit heaters the steam piping must be size to carry a full supply of steam to the unit to take the place of that condensed. In hot water systems, piping should be sized properly to handle required flow (GPM) of water.

When installing steam unit heaters the supply line should be pitched towards the steam main in order to prevent condensate in the main from draining through the heater, where it might reduce capacity and cause noise. With hot water unit heater both supply and return branch lines should be pitched towards the unit.

The return line from steam unit heater should provide a minimum drop of 300 mm (12") below

The heater, so that the heat of water required to overcome the resistances of check valves, traps, and strainers will not cause condensate to remain in the coil.

4- Rapid condensation of steam, specially during heating-up periods, is characteristic of steam unit heaters. Steam traps with ample capacity to handle the condensate when the unit is operating under maximum steam pressure and minimum entering air temperature plus a suitable safety factor should be selected.

5- Dirt pockets at the outlet of unit heaters are essential, and strainers with 1.5mm (1/16") perforations are recommended as additional means of retaining dirt and scale which might affect operation of check valves and traps.

6- The vertical connection to the air vent should be at least 3/4" pipe thread, to permit separation of the water from the air passing to the vent.

7- To ensure optimum performance of the unit heaters at the rated capacity, all supply mains and branch connections to the units should be insulated.

8- Steam piping and unit heaters should be supported independently.

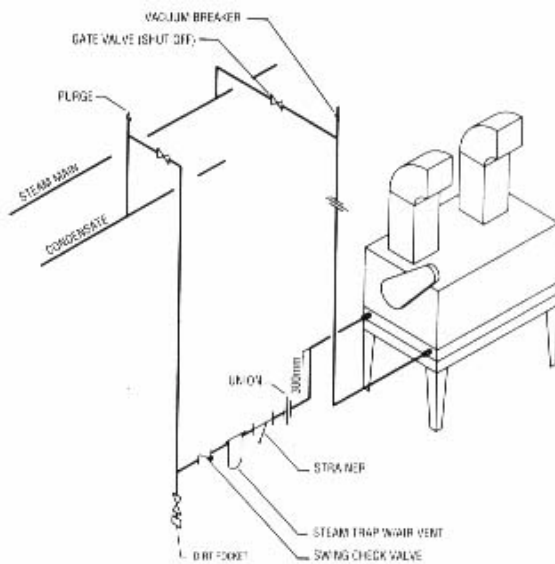


FIGURE 3. STEAM HEATING SYSTEMS

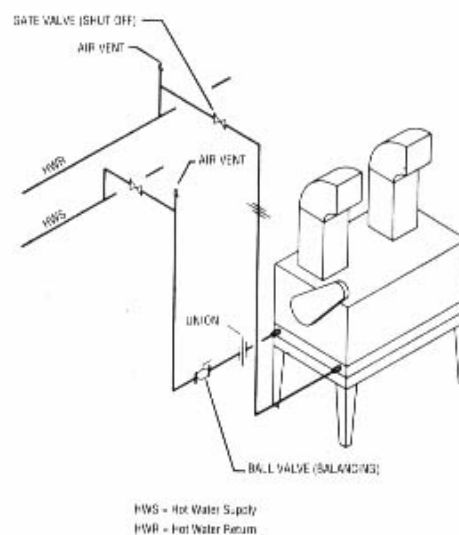


FIGURE 4. HOT WATER SYSTEMS

**UNIT CASINGS**

All casings shall be constructed of heavy gage galvanized steel sheets painted with hammer finish after final assembly.

COILS

Coils shall be constructed of galvanized steel sheet casings with copper tubes or (seamless steel tubes). For hot water applications with temperature exceeding 130°C and high pressure steam applications greater than 30 psig, 1/2" seamless steel tubes shall be utilized with spiral fins made of aluminium or copper. For hot water and steam applications under normal working temperature and pressure, 5/8" copper tubes with aluminium (copper) fins in plate form shall be used. Fin spacing shall be 8FPI. Steam coils shall be 1 or 2 tubes rows, and hot water coils shall be 2 or 3 tubes rows.

All coils shall be pressure tested with 325 psig air in an illuminated tank according to ASHRAE Standard 15-1992 (Safety Code for Mechanical Refrigeration).

FANS

All fans shall be of double inlet, forward curved centrifugal type fabricated from galvanized sheet metal. Fan assembly including pulleys, sheaves, and shafts shall be statically and dynamically balanced.

All shafts shall be oxide conversion coated.

MOTORS

All motors shall be 3 phase-380-50Hz and shall operate at 1450 RPM or 950 RPM (for applications where low noise levels are important). Motors with spark proof construction shall also be employed for applications where flammable and volatile gases might leak into the area.

DAMPERS

Standard face and bypass dampers shall be of parallel blade type with interconnecting linkage. Blade bearings shall be plastic bushings and shall provide smooth operation and corrosion resistance.



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